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Optimized Breech Location in the Harry Diamond Laboratories
4-Inch Gas Gun

Herbert D. Curchack

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20. ABSTRACT (Cont'd)

theoretical predictions are compared for several gun divisions, helium and nitrogen drivers, and initial accelerations from 200 to 5000 g. For improved gun utilization, gun configurations are suggested to reduce exit gas pressure, to reduce the quantity of driver gas required, and to ease operation and improve (statistical) utilization of the gun without unduly affecting muzzle velocity. Although the results obtained are specific to the Harry Diamond Laboratories 4-in. gas gun, computer programs are included that may be applied to other gun geometries.

CONTENTS

	<u>Page</u>
1. BACKGROUND.....	5
2. CONSTRUCTION.....	6
3. VELOCITY MEASURING INSTRUMENTATION.....	9
4. OTHER MEASUREMENTS.....	9
5. THEORY.....	10
5.1 Isentropic Expansion.....	10
5.2 Computer Analyses.....	11
5.3 Pidduck-Kent Special Solution.....	11
5.4 Constant Diameter, Infinite Length Driver Theory.....	14
5.5 Combined Pidduck-Kent and Infinite Driver Theories.....	17
6. EXPERIMENTAL RESULTS.....	20
6.1 Procedure.....	20
6.2 Data.....	20
7. DISCUSSION.....	24
8. CONCLUSIONS AND REMARKS.....	28
ACKNOWLEDGEMENT.....	29
NOMENCLATURE.....	29
APPENDIX A.—COMPUTER PROGRAMS.....	31
DISTRIBUTION.....	47

FIGURES

1. Four-inch gun.....	5
2. Four locations for release pin of constant length gun.....	5
3. Muzzle section of 4-in. gun.....	6
4. One-atmosphere configuration breech of 4-in. gun.....	7
5. Closure section.....	7
6. Projectile restraining section.....	8
7. Isometric side view of 4-in. gun.....	9

FIGURES (Cont'd)

	<u>Page</u>
8. Pidduck-Kent theory, helium, 200 to 3000 g.....	12
9. Pidduck-Kent theory, nitrogen, 200 to 3000 g.....	12
10. Pidduck-Kent theory, helium, 3000 to 15,000 g.....	13
11. Pidduck-Kent theory, nitrogen, 3000 to 15,000 g.....	14
12. Infinite driver theory, helium, 200 to 3000 g.....	15
13. Infinite driver theory, nitrogen, 200 to 3000 g.....	16
14. Infinite driver theory, helium, 3000 to 15,000 g.....	16
15. Infinite driver theory, nitrogen, 3000 to 15,000 g.....	17
16. Pidduck-Kent and infinite driver theories combined, helium, 200 to 3000 g.....	18
17. Pidduck-Kent and infinite driver theories combined, nitrogen, 200 to 3000 g.....	19
18. Pidduck-Kent and infinite driver theories combined, helium, 3000 to 15,000 g.....	19
19. Pidduck-Kent and infinite driver theories combined, nitrogen, 3000 to 15,000 g.....	20
20. Typical projectiles used in study.....	21
21. Data and theories, nitrogen, relative pin position of 0.117.....	22
22. Data and theories, nitrogen, relative pin position of 0.372.....	23
23. Data and theories, helium, relative pin position of 0.372.....	23
24. Combined data, nitrogen and helium, relative pin positions of 0.117 and 0.372.....	24
25. Lagrange and Pidduck-Kent theories, helium, relative pin positions of 0.117, 0.244, and 0.372.....	25
26. Helium (Lagrange theory).....	26
27. Nitrogen (Lagrange theory).....	26
28. Fraction of energy gained by projectile from driver gas (Lagrange theory).....	27

TABLES

1. Features of Optimized Lagrange Gun.....	11
2. Effects of Projectile Restraining Section With Helium Driver.....	28

1. BACKGROUND

Gas guns are used at the Harry Diamond Laboratories (HDL) to accelerate ordnance items to velocities typical of mortar, recoilless rifles, and artillery projectiles, that is, 400 to 4000 ft/s (120 to 1200 m/s). Test considerations often dictate achievement of the required velocity with the lowest acceleration (g) force exerted on the projectile while in the gun. This dictum necessitates the use of low driving pressures, and therefore long guns are needed to achieve the required velocities. These guns are installed in special purpose laboratory space, and the maximum gun length is limited by the dimensions of the building. The characteristics of the guns and the governing theories must be known to configure the guns so as to maximize the projectile velocity for the applied acceleration.

The gun used most often at HDL has a 4-in. bore and is the subject of this experimental and theoretical investigation. A large portion of the gun is located in a 5-ft (1.5-m)-high tunnel that runs the width of the building that it occupies. Several meters of each end of the gun extend into a "breach" room and a "muzzle" room. This gun can be configured in two ways (fig. 1). For many tests, velocities of 400 to 800 ft/s (120 to 240 m/s) are adequate. These are readily achievable in a 1-atm configuration. In the 1-atm gun, a pin restrained projectile seals one end of a barrel, a diaphragm seals the other end, and the barrel is evacuated (fig. 1a). Removal of the pin per-

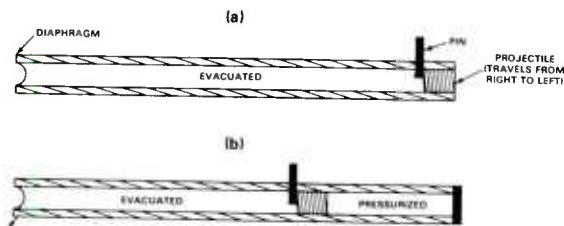


Figure 1. Four-inch gun: (a) one-atmosphere and (b) pressurized configurations.

mits atmospheric pressure to accelerate the projectile down the barrel and through the diaphragm. When tests require higher velocities, a pressurized driver is required. The gun contains provision for a second pin downstream from the 1-atm pin. The breech end of the gun is sealed, and the section between this end of the gun and the projectile can be pressurized (fig. 1b). This configuration uses part of the gun length as driver and therefore shortens the length of the gun available as barrel.

Consider the four pressure-driven gun configurations shown in figure 2. In the uppermost gun (fig. 2a), the driver gas has no volume, and therefore the projectile can achieve no velocity. The projectile moves with increasing speed as the pin is located farther downstream (fig. 2b). Further movement of the pin (fig. 2c) shortens the barrel length (the distance over which the projectile accelerates) and thus offsets the gain in chamber volume. Finally, as the pin location nears the muzzle end of the gun (fig. 2d), the projectile has little distance over which to accelerate, and the muzzle velocity approaches zero.

The problem is to locate the pin at the position that, for any specified velocity, requires the lowest gas pressure. This is equivalently stated as the pin position that maximizes the projectile muzzle velocity for a given gas pressure. There are a limited number of pin

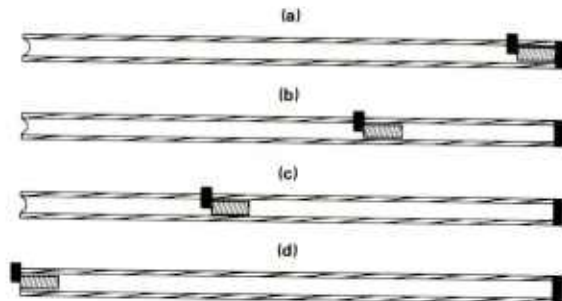


Figure 2. Four locations for release pin of constant length gun: (a) at breech, (b) near breech, (c) near muzzle, and (d) at muzzle.

positions (breech locations) in the 4-in. gun because of the way that the gun is constructed. This report shows that no one pin position is optimal for all velocities and, because it is impractical to relocate the breech for different tests, the solution to the problem becomes somewhat subjective.

2. CONSTRUCTION

The gun used in this investigation is constructed from eight 12-ft (4-m) lengths of 90-mm gun barrels. The barrels have been bored to a 4-in. (10-cm) inside diameter (i.d.), turned down to a 6-in. (15-cm) outside diameter (o.d.), and flanged. The barrel sections are always evacuated, even when a pressure driver is used. The joints are sealed with copper gaskets. The muzzle section of the gun (fig. 3) attaches to the last barrel and contains the fittings for the vacuum connections. This section has a large internal toroidal reservoir to accept any gas that is driven ahead of the projectile and thereby to minimize the pressure rise ahead of the projectile. This section contains also the muzzle seal, which is a 0.002-in.

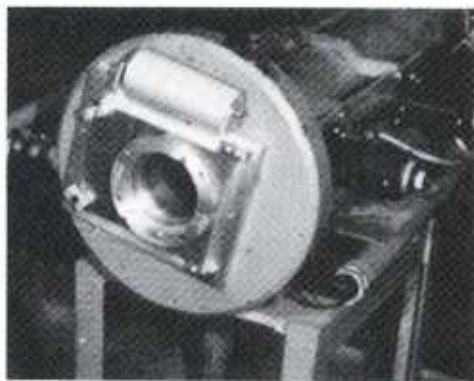


Figure 3. Muzzle section of 4-in. gun.

(50- μ m) Mylar sheet clamped in a door by two screw latches. The Mylar is located on a roller above the door, and a new section of the sheet is pulled down and reclamped after each shot.

The release pin assembly used in the 1-atm configuration (fig. 4) attaches to the entrance of the first barrel. It is a short gun extension with an internal circumferential O-ring groove to seal to the girth of the projectile and a pneumatically operated, electrically controlled pin to restrain the projectile. Also visible is a backstop, which acts as a backstop to capture the projectile if it is propelled out of the rear of the gun in the event that the muzzle diaphragm is accidentally ruptured. To the left of the release pin assembly is a side access port, which remains covered in the 1-atm configuration and is used only in the pressurized configuration.

In the pressurized configuration, the breech end of the gun is sealed by a closure section located in the side access port in the first barrel section (fig. 5). The closure section contains pressurization and vacuum ports.

The projectile restraining section used in the pressurized configuration (fig. 6) is 1 ft (30 cm) long; has a side access panel 8 in. long; and contains a pneumatically operated, electrically controlled pin and the required O-ring. The projectile restraining section can be placed between any two sections of the gun and therefore can be placed at one of seven possible locations along the barrel. When the 1-atm configuration is used, the projectile restraining section O-ring is removed, and the pin is withdrawn to allow free projectile passage through the projectile restraining section.

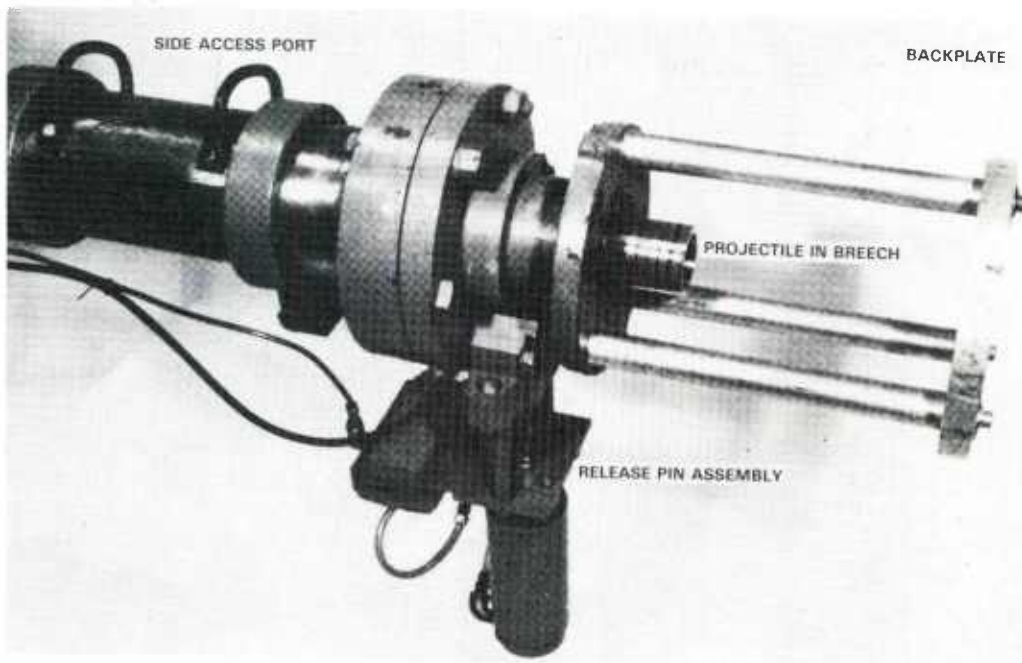


Figure 4. One-atmosphere configuration breech of 4-in. gun.

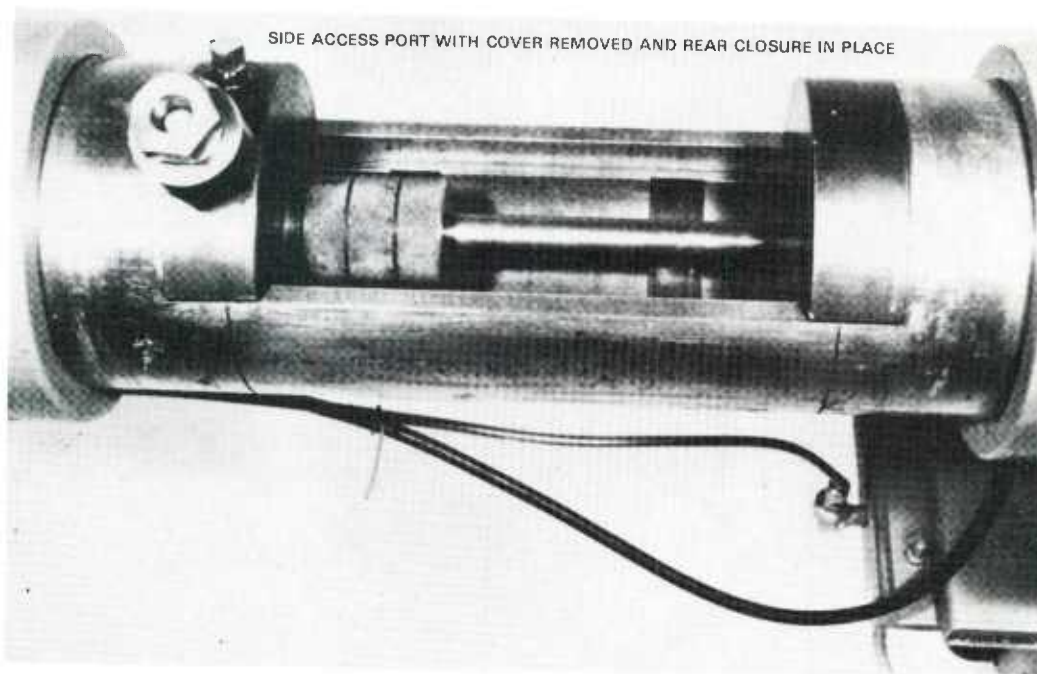


Figure 5. Closure section.

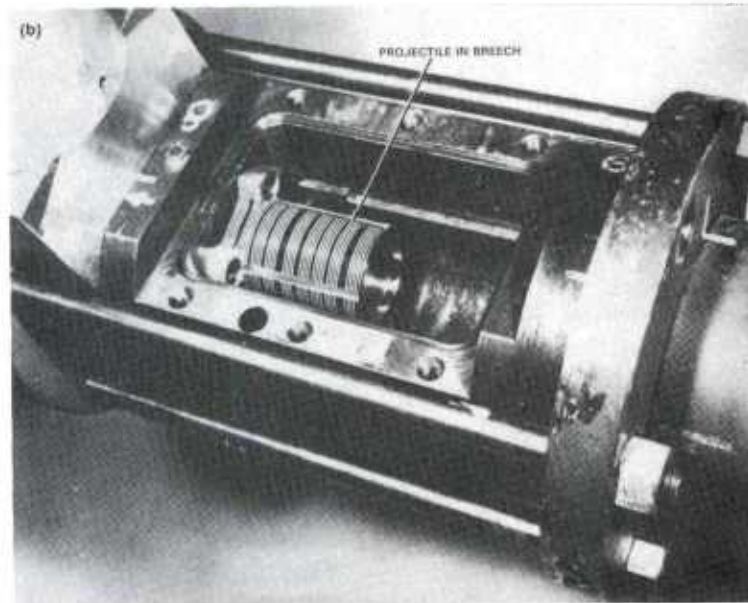
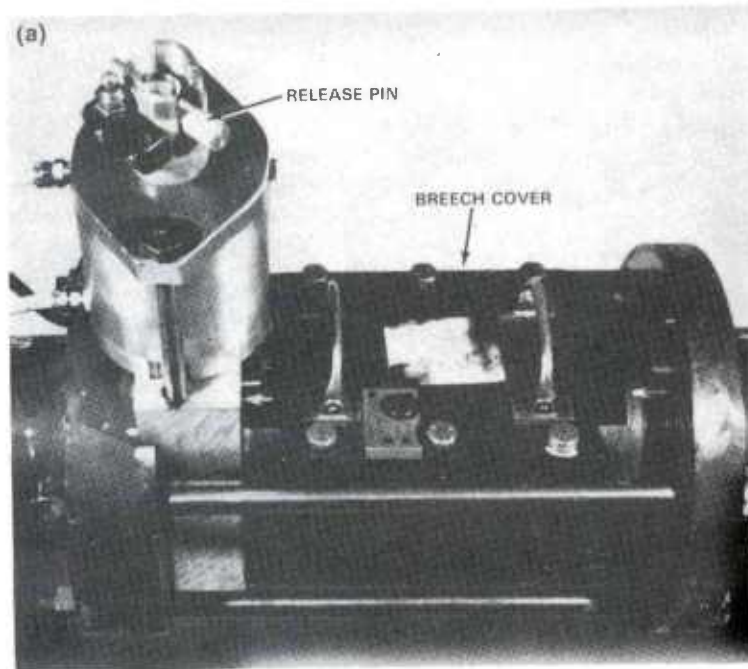


Figure 6. Projectile restraining section: (a) open and (b) closed.

The drawing in figure 7 shows the 4-in. gun with the projectile restraining section mounted between the first two barrel sections. This assembly provides easy access to the projectile restraining section because it is located in the breech room. Currently, the projectile restraining section is mounted between barrel sections 3 and 4 and is located in the gun tunnel. This location is somewhat inconvenient because of the low ceiling.

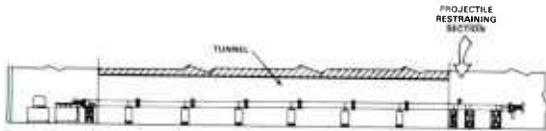


Figure 7. Isometric side view of 4-in. gun.

3. VELOCITY MEASURING INSTRUMENTATION

Muzzle speed is obtained by measurement of the time of flight between two "light screens" located 12.00 in. (30.48 cm) apart near the end of the last gun barrel section. All the experimental projectile muzzle velocity data presented below are computed by using these devices. For theoretical purposes, the end of the gun is assumed to be halfway between the two light screens.

Each light screen consists of a light source operated by direct current (dc), a photodetector, and an amplifier circuit. The source and the detector face each other along a diameter of the gun tube. When the projectile interrupts the light falling on the detector, a trigger signal is generated.

The light source consists of an incandescent lamp mounted behind an achromatic lens, which images the lamp filament about 200 mm in front of the lens. A lens in the detector assembly refocuses the light onto a small piece of frosted glass positioned immediately in front of the photodetector. With this arrangement, the photodetector amplifier output jumps from 2 to 18 V when the light beam is inter-

rupted. The rise time of each detector and amplifier combination is 15 μ s, with less than 1- μ s difference between the two light screens. Therefore, time interval measurements are accurate to within 1 μ s; positioning of the light screens is accurate to within 0.01 in. (0.25 cm); hence, the accuracy of the velocity measurements is approximately 0.4 ft/s (12 cm/s) at 400 ft/s (120 m/s) and about 6 ft/s (1.8 m/s) at 2500 ft/s (750 m/s).

Previous setups with finer beams and therefore finer distance resolution proved to be sensitive to debris or vibrations or both and occasionally triggered early. The present arrangement has proven to be highly reliable.

Streak photography, which is used to photograph the projectile shortly after it leaves the muzzle, provides a further check on muzzle velocity. The streak method (used for muzzle velocities less than 1500 ft/s or 300 m/s) agrees with the above method to better than 1 percent.

4. OTHER MEASUREMENTS

Other measurements that enter into the calculation of the initial acceleration are projectile weight, gas pressure, gun length, and gun diameter. By far, the least accurate of these measurements is the gas pressure, which is good to only 5 percent; hence, measurements of the initial acceleration are good to about 5 percent.

Certain gas parameters are used in the theory. The gas properties assumed are the properties of a pure gas at 20 C. The room (driver) temperature was constant to about 3 C, which has little effect on these properties. For nitrogen experiments, no special care was taken to assure driver gas purity. It was assumed that the air initially in the driver section was sufficiently close in properties to nitrogen so as not to influence the results. For helium shots, a purging technique was implemented

consisting of three series of evacuation of the driver section to less than 5 psia (34.5 kPa absolute) followed by helium pressurizations to 50 psig (345 kPa gauge) (except for the final pressurization, which was to the required firing pressure).

5. THEORY

The problem of optimization of pin position in the 4-in. gun, along with a more general problem that includes diameter changes in the driver section, has been analyzed by Seigel.¹ Seigel's report contains an analysis of the four active HDL gas guns and two future guns. The aim of his investigation is to configure the guns so as to maximize the projectile velocity for a specified maximum projectile acceleration. The aim of HDL's investigation is stated in section 1.

5.1 Isentropic Expansion

It is instructive to start the theoretical discussion with a problem that is analytically tractable, adds insight to the results, and is shown to be valid for certain test conditions. Isentropic expansion of a driver gas² yields an approximate prediction of gun muzzle velocity (Lagrange approximation) when the gas sound speed is much larger than the projectile velocity (U), that is, small Mach number. It is assumed that the driver section diameter is the same as the barrel diameter.

$$U^2 = K [NZ (1 - Z^{2/N})/2] , \quad (1)$$

where

$$K = 2APL/M , \quad (1a)$$

$$Z = X/L , \quad (1b)$$

N is the degrees of freedom of the driver gas,

¹Arnold E. Seigel, *Performance Calculations and Optimization of Gas Guns*, Chevy Chase, MD, HDL-CR-81-723-1 (May 1981).

²J. Corner, *Theory of the Interior Ballistics of Guns*, John Wiley & Sons, New York (1950).

A is the cross-sectional area of the gun barrel, P is the initial gas pressure in the driver reservoir, L is the total gun length, M is the projectile mass, Z is the relative position of the pin measured from the breech end of the barrel, and X is the length of the driver reservoir.

The square root of K is the velocity (U_{total}) that would be achieved in a constant acceleration gun of the total gun length.

$$U_{total} = K^{1/2} . \quad (1c)$$

The quantity in brackets in equation (1) is always less than 1 and therefore represents a decrease from U_{total} .

Since the ratio of barrel length (Y) to total gun length is $1 - Z$,

$$U_{barrel} = U_{total} (1 - Z)^{1/2} . \quad (1d)$$

U_{barrel} is the velocity that would be attained if the acceleration were constant over the barrel length.

The optimum pin position (Z_{opt}) is found by differentiating U with respect to Z and setting the result to zero. These actions result in

$$Z_{opt} = (1 + 2/N)^{-N/2} , \quad (2)$$

$$U_{max} = U_{total} [Z_{opt} / (1 + 2/N)]^{1/2} , \quad (3)$$

$$U_{max} = U_{barrel} [N/(N + 2)]^{1/2} \times [Z_{opt} / (1 - Z_{opt})]^{1/2} . \quad (4)$$

Table 1 lists results from equations (2) to (4). Note these:

- a. The greater the degrees of freedom of the gas, the shorter the driver and the higher the expected velocity.

b. When the pin is located about 40 percent of the way down the length of the gun barrel, we should expect about 55 percent of the velocity achievable if the entire gun length were available for constant acceleration.

c. Velocity based on the actual barrel length indicates that the projectile can never achieve better than three-quarters of the constant acceleration velocity.

TABLE 1. FEATURES OF OPTIMIZED LAGRANGE GUN

N	Z _{opt}	U _{max} /U _{total}	U _{max} /U _{barrel}
3	0.465	0.528	0.722
5	0.431	0.555	0.736
Infinite	0.368	0.606	0.763

5.2 Computer Analyses

Theories that account for inertial effects of the gas were programmed in VAX-11 BASIC and are treated in the following sections. The programs are included in appendix A and are referred to as used in this paper. For ease of presentation, the results are in

graphical form. The results obtained, although sometimes presented as dimensionless numbers, are not generally applicable. Guns of different lengths yield different curves as functions of the same dimensionless numbers. Such curves can be obtained only by changing the initial conditions in the programs.

5.3 Pidduck-Kent Special Solution

An important analytic treatment is the Pidduck-Kent solution, which assumes that through a number of reflections between the rear of the driver and the projectile, the flow has reached a steady pressure profile, which decreases in amplitude with time. The equations governing this solution are included in the program PDQ and are not repeated here.

In the following graphs, projectile velocity is presented as a function of relative pin position, that is, driver section length per total gun length. The parameter is the initial acceleration, which is the same as the maximum acceleration. Figure 8 presents results for a helium driver. The initial acceleration is 200 g for the lowest curve and 3000 g for the highest curve, and intermediate curves are in 200-g increments. Figure 9 shows the same results for nitrogen.

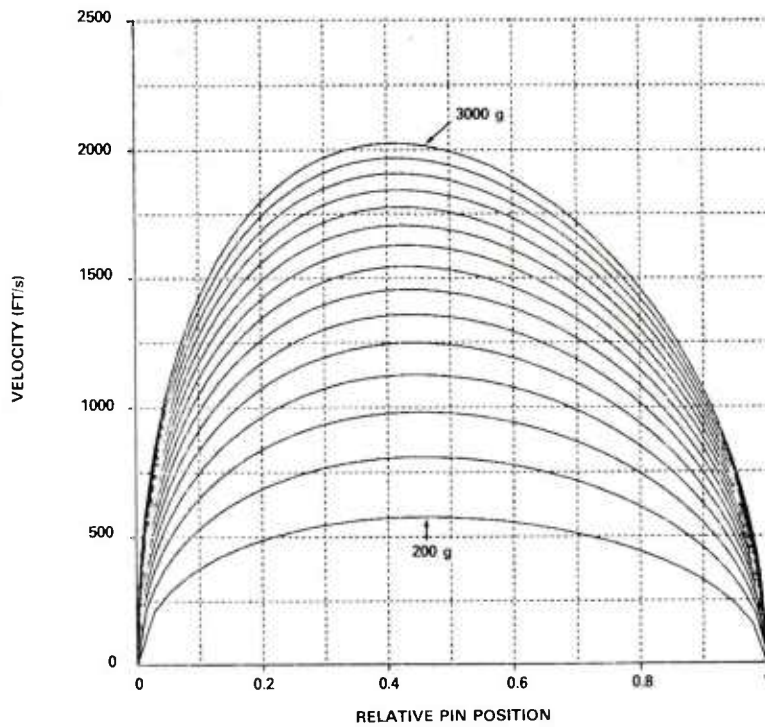


Figure 8. Pidduck-Kent theory, helium, 200 to 3000 g.

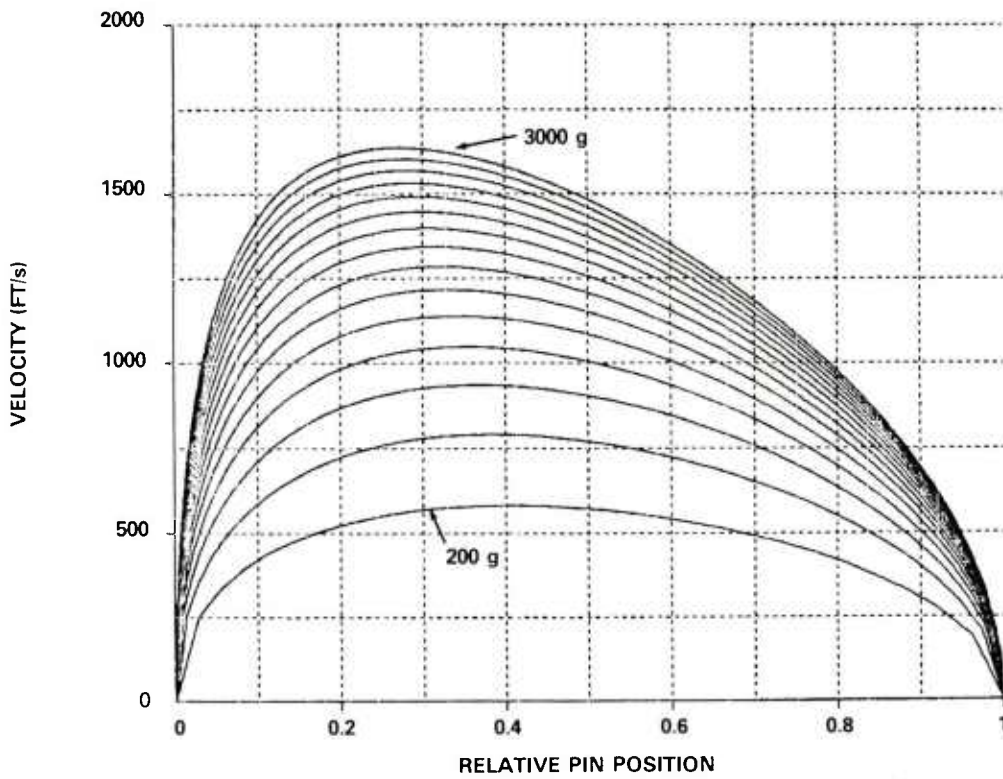


Figure 9. Pidduck-Kent theory, nitrogen, 200 to 3000 g.

Note that the relative pin position for the velocity peaks for the lowest curves in each figure, 0.467 and 0.417, fall near those predicted by the Lagrange theory in table 1. However, as the initial acceleration increases, the peaks shift to smaller values of relative pin position. The better agreement for the helium data is attributed to the assumption that a weightless gas, inherent in the Lagrange theory (which postulates infinite sound speed), is better satisfied; the Mach number for the 200-g helium case is only 0.15, but for the nitrogen case it is up to 0.5. The lowest curves (200 g) also predict velocities of 576 and 582 ft/s (173 and 175 m/s) for helium and nitrogen,

that is, higher for nitrogen! However, as the initial acceleration increases, the higher sound speed of helium becomes of significance, and higher projectile velocities are obtained with the monatomic gas.

Figures 10 and 11 present data for initial accelerations of 3000 to 15,000 g in steps of 2000 g. Although accelerations of more than 3000 g are rarely anticipated for the particular application of this investigation,³ these results are included for completeness.

³Herbert D. Curchack, *An Artillery Simulator for Fuze Evaluation*, Harry Diamond Laboratories HDL-TR-1330 (November 1966).

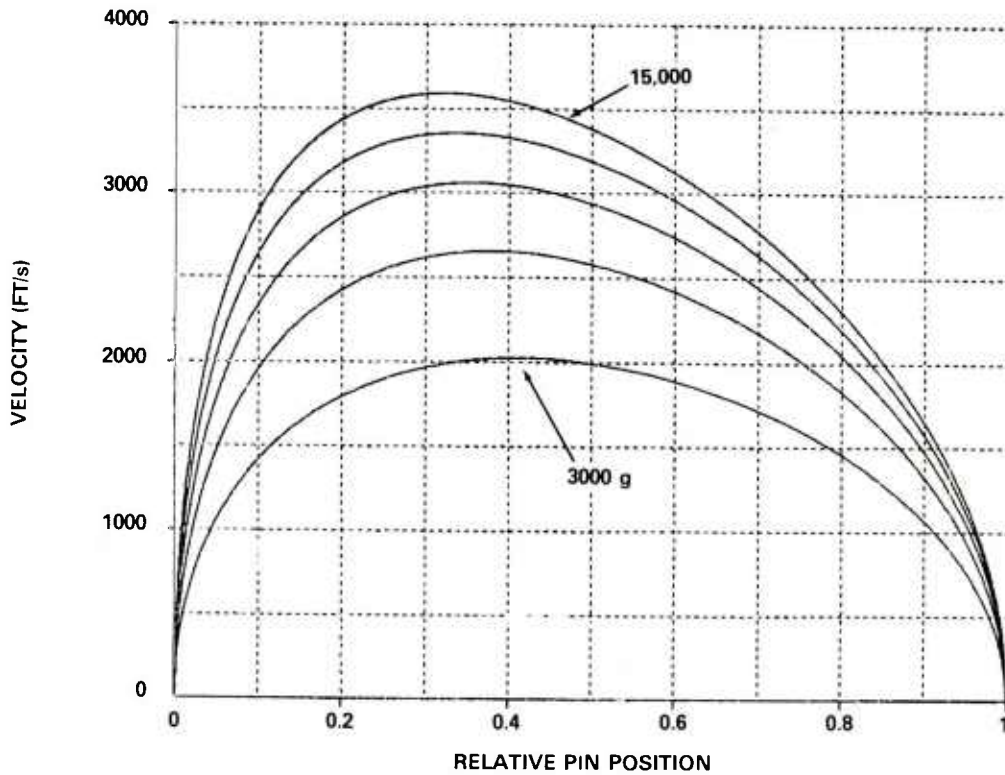


Figure 10. Pidduck-Kent theory, helium, 3000 to 15,000 g.

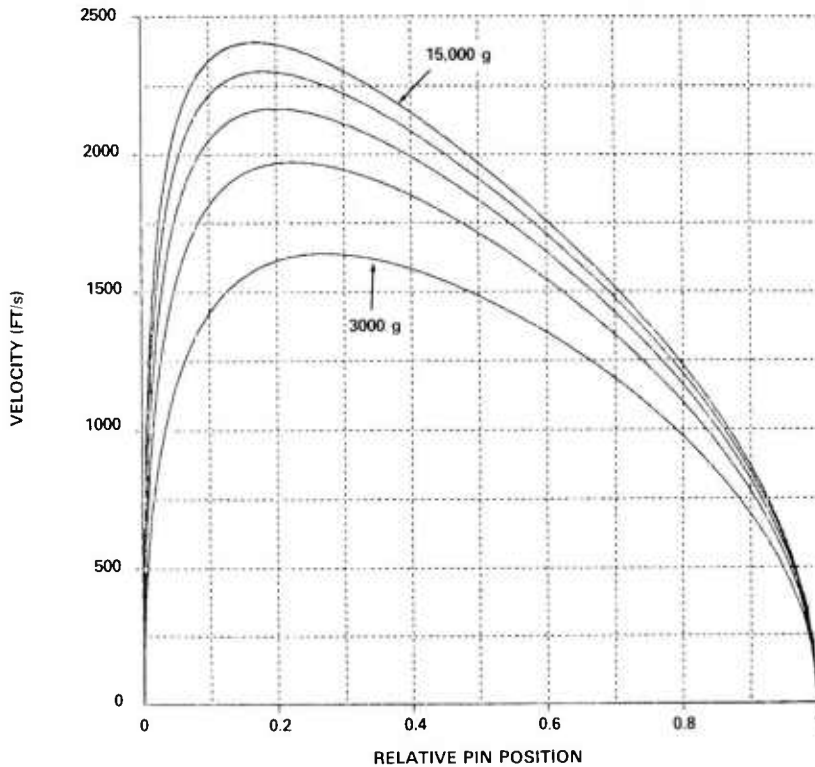


Figure 11. Pidduck-Kent theory, nitrogen, 3000 to 15,000 g.

5.4 Constant Diameter, Infinite Length Driver Theory

If the driver is assumed to be the same diameter as the barrel and infinite in length,⁴ then program CONDIA is applicable. Although the governing equation can be deduced from the program, it is camouflaged by changes in notation and is repeated here.

$$\text{PAY}/\text{Ma}^2 = [u - N/(N + 1)](1 - u/N)^{(N + 1)} + N/(N + 1), \quad (5)$$

where a is the initial sound speed in the driver reservoir and u is the projectile Mach number based on initial sound speed. This theory applies where the driver "appears" infinite. That

⁴Arnold E. Seigel, *The Theory of High Speed Guns*, Naval Ordnance Laboratory, White Oak, MD, AGARDograph 91 (May 1965), eq (12.2).

is, when the first expansion wave (of an expansion fan) leaves the projectile at the onset of motion, the wave is reflected from the rear of the gun, but does not overtake the projectile before the projectile exits from the muzzle. The resultant curves are plotted in figures 12 to 15 for helium and nitrogen and for the lower and higher initial acceleration ranges.

The diamonds on the curves represent the projectile starting location for which the first reflected wave overtakes the projectile just as it leaves the muzzle. They were obtained from program CATCH. The assumption of infinite length driver is valid for all points to the right of the diamonds, and the validity decreases to the left.

These curves do not have theoretical maxima as in the Pidduck-Kent and Lagrange

cases. The optimum pin location from these curves is at a relative pin position of zero ($Y = L$), which is in a region in which the theory does not hold. (The driver is of zero length, not infinite length.) Therefore, a determination of optimum pin position from these curves is not possible. However, the first waves that over-

take the projectile are weak expansion waves, and theoretical results that include effects of these waves would follow infinite driver results for some distance to the left of the diamond. Based on this consideration, the curves would continue to rise for some distance to the left of the diamonds, and optimum relative pin position would be to the left of the diamonds.

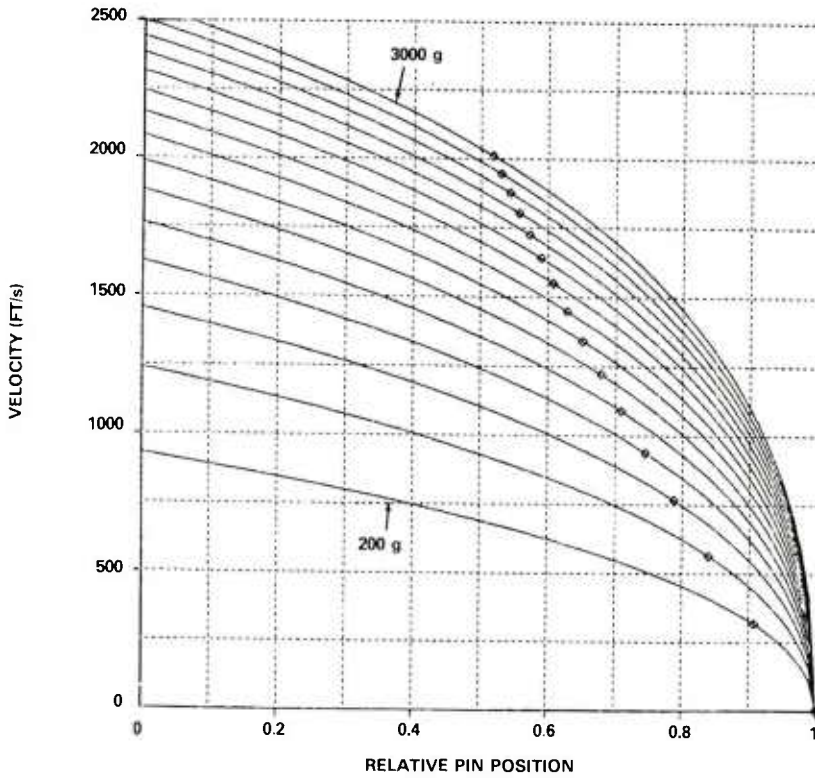


Figure 12. Infinite driver theory, helium, 200 to 3000 g.

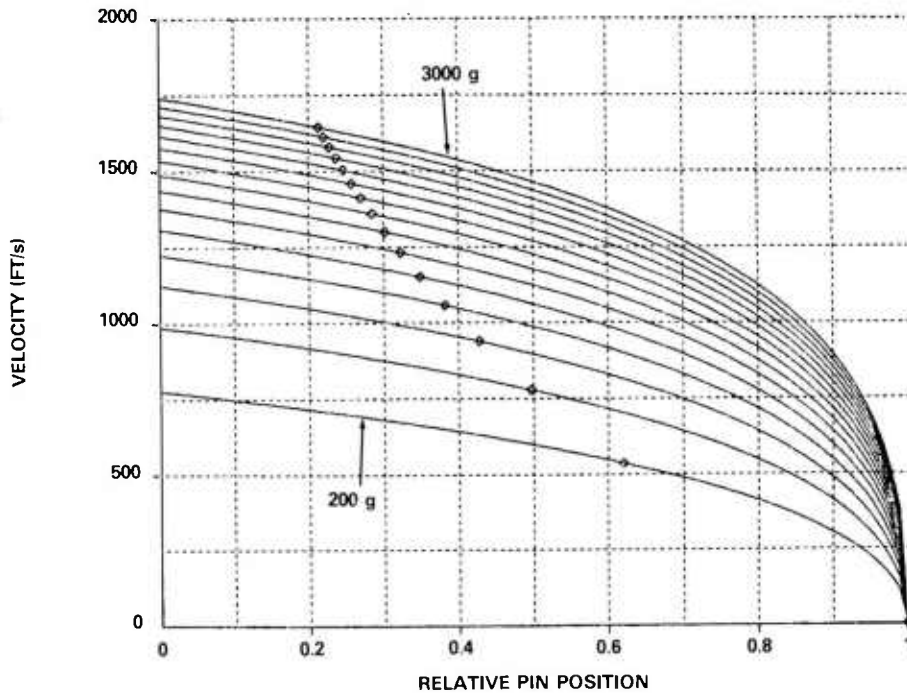


Figure 13. Infinite driver theory, nitrogen, 200 to 3000 g.

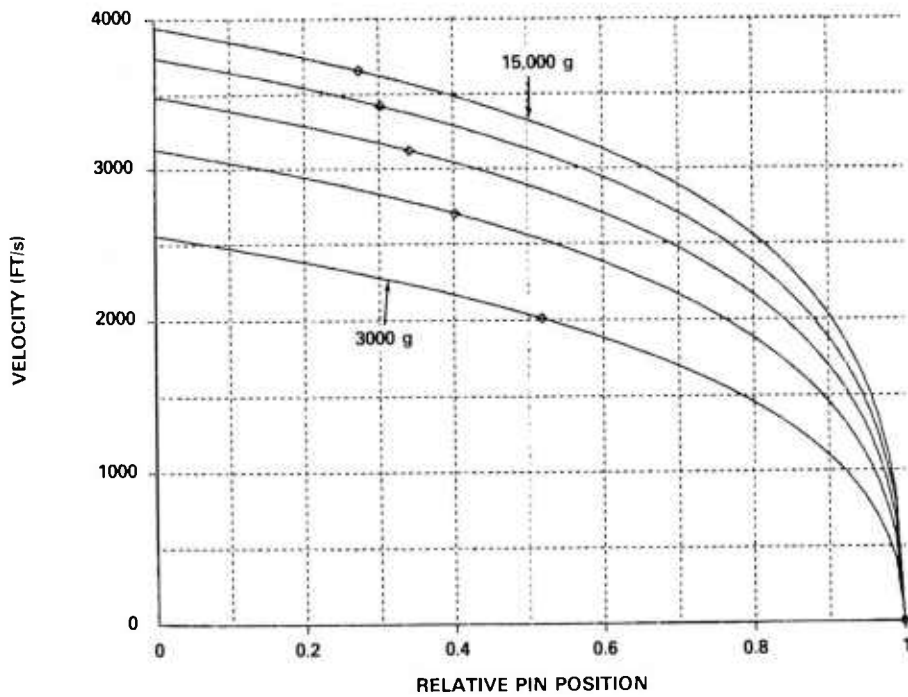


Figure 14. Infinite driver theory, helium, 3000 to 15,000 g.

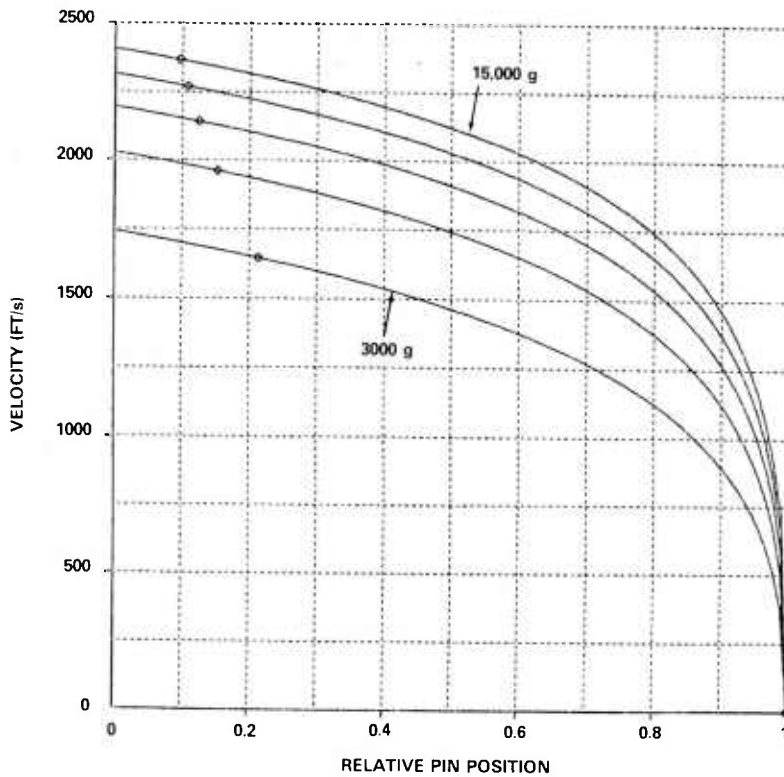


Figure 15. Infinite driver theory, nitrogen, 3000 to 15,000 g.

5.5 Combined Pidduck-Kent and Infinite Driver Theories

The results presented in figures 16 to 19 were obtained from program CDPK, which combines the results of Pidduck-Kent for small values of relative pin position with infinite driver results for large values. These were obtained by comparing the predicted velocities from the two theories starting at a relative pin position of zero and using the Pidduck-Kent result as output so long as it predicted a velocity lower than the infinite driver results. Once the infinite driver theory predicted lower velocity, infinite driver results were used for the remainder of the run, even though for higher values of relative pin position Pidduck-Kent again predicted lower muzzle velocity. Infinite driver results were used because infinite driver theory is more appropriate in this region. The

point at which the two theories intersect is marked with a plus sign. The diamond represents the projectile starting location for which the first reflected wave overtakes the projectile just as it leaves the muzzle.

Generally, for low initial accelerations, the transition from one theory to the other is smooth, and the first reflection points are on or very close to the Pidduck-Kent portion of the curve. For higher accelerations, the transition becomes more abrupt, and the first reflection points lie above the Pidduck-Kent curves. Since infinite driver theory (sect. 5.4) implies that the maximum is to the left of the diamond, the conclusion drawn from these figures is that Pidduck-Kent underestimates muzzle velocity in the transition region, that is, where the first reflection overtakes the projectile near the muzzle. Since Pidduck-Kent

assumes a gas pressure distribution that is achievable only after several reflections and infinite driver assumes no reflections from the projectile, this divergence is not unexpected. To use these graphs to predict optimum relative pin position now becomes difficult, especially for nitrogen. Although the curves have well-defined peaks, these peaks are in the region where the wrong theory defines the peaks. To reiterate, the "real" peak must be someplace to the left of the diamonds.

Another consideration in predicting velocity is associated with the very low Mach number region. For example, consider low initial accelerations with helium. The sound speed is sufficiently high so that many wave reflections occur while the projectile moves only slightly. This situation leads to a nearly

uniform gas pressure between the projectile and the end of the breech. When the projectile remains at a low Mach number, the pressure drops uniformly in the driver section as the projectile moves and the volume increases. Under this condition, Lagrange results should be more appropriate than Pidduck-Kent results.

The net result of these theoretical considerations is that the velocity peak moves toward lower relative pin position with increasing initial acceleration, the peak is somewhere to the left of and near the diamonds, and the peaks differ in amplitude and location for helium and nitrogen. However, experimental results must be evaluated to verify the appropriateness and the range of any of the theories presented.

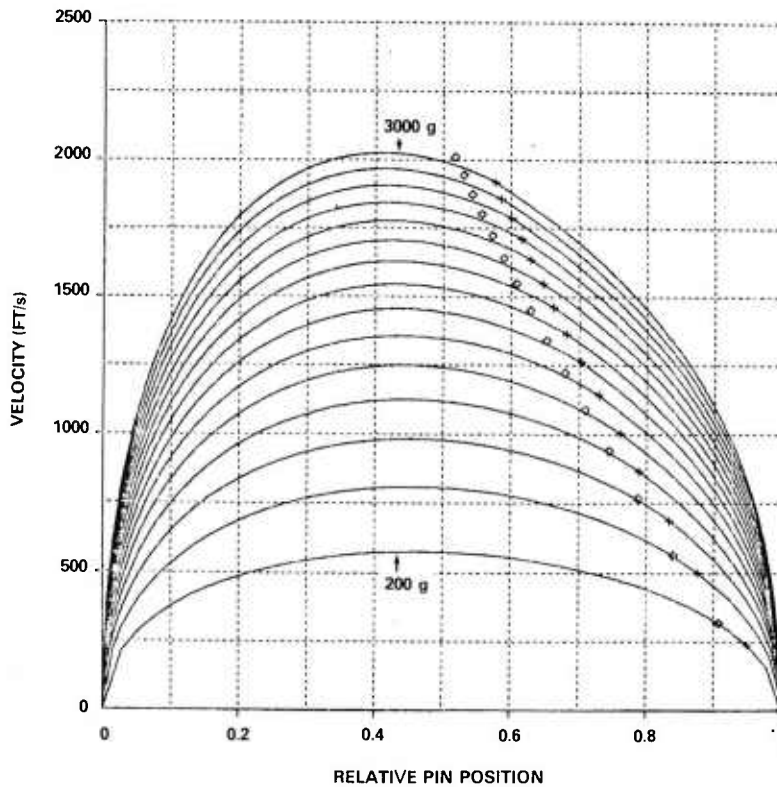


Figure 16. Pidduck-Kent and infinite driver theories combined, helium, 200 to 3000 g.

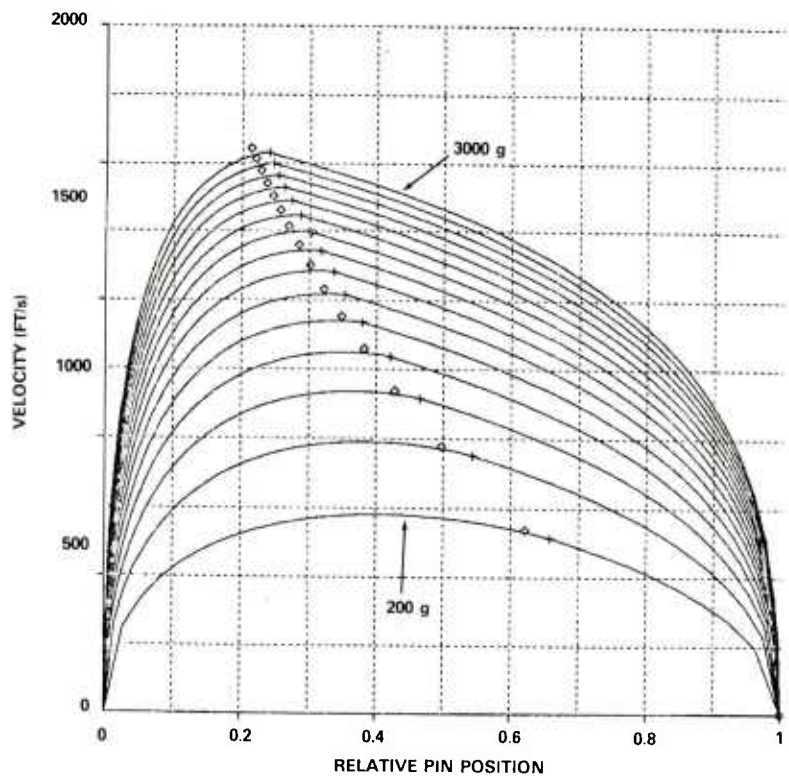


Figure 17. Pidduck-Kent and infinite driver theories combined, nitrogen, 200 to 3000 g.

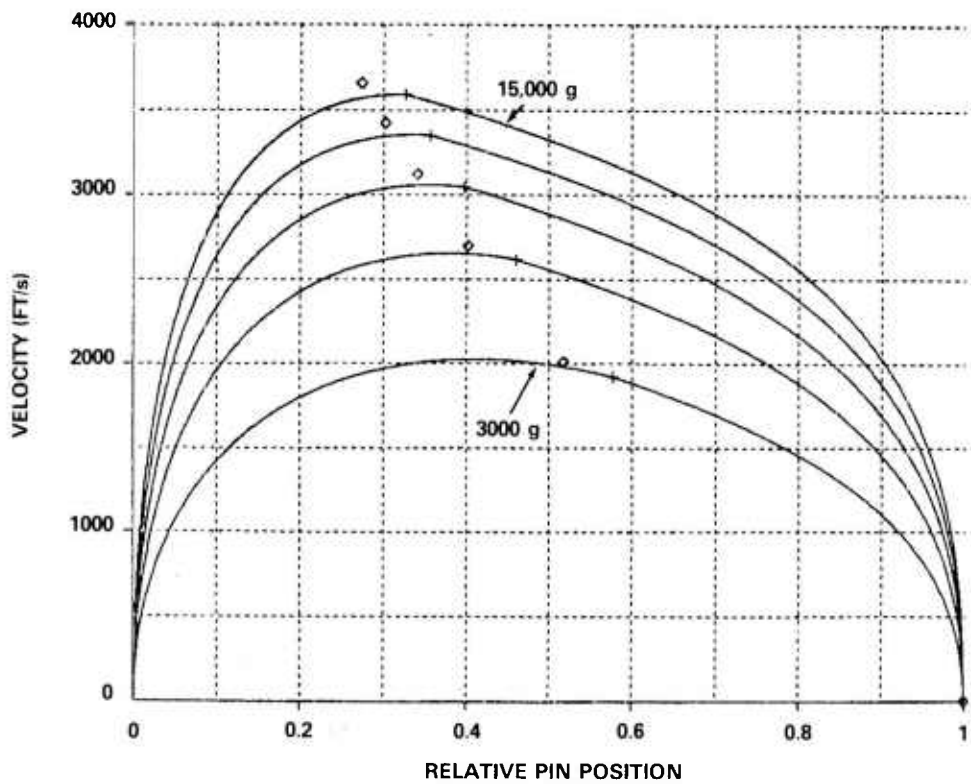


Figure 18. Pidduck-Kent and infinite driver theories combined, helium, 3000 to 15,000 g.

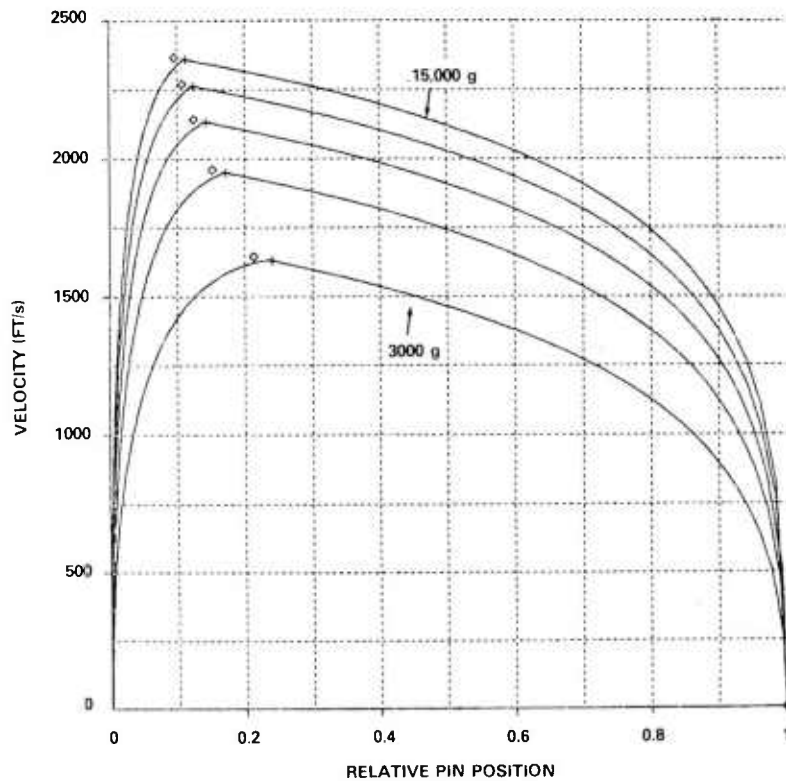


Figure 19. Pidduck-Kent and infinite driver theories combined, nitrogen, 3000 to 15,000 g.

6. EXPERIMENTAL RESULTS

6.1 Procedure

Many of the results included in what follows were made during routine operation of the gun. The projectiles were not specially prepared for optimum gun performance (fig. 20). Several projectiles were used. The most critical tolerances for these experiments, those associated with projectile diameter and the distance between the two bore riders (wheel base), were only grossly controlled. The projectile diameter ranged from 0.004 to 0.015 in. (102 to 381 μm) less than the gun i.d. The wheel base was typically 1 to 1.5 calibers. The center of mass of the projectile typically was near or aft of the rear bore rider.

6.2 Data

Experiments have been run with nitrogen with the projectile restraining section at two positions and with helium with the projectile restraining section at one of these positions. Maximum pressure was 300 psia (2070 kPa absolute). Projectile weights varied from 0.5 to 5 lb (0.25 to 2.5 kg) and covered the velocity range from 500 to 2600 ft/s (150 to 780 m/s). The first series of experiments had the projectile restraining section at the end of the first breech section; this was the easiest section to access. Sixty-nine shots were fired using nitrogen with the projectile restraining section located here. Subsequently, 229 tests with nitrogen and 94 tests with helium were run with the projectile restraining section between gun sections 3 and 4.

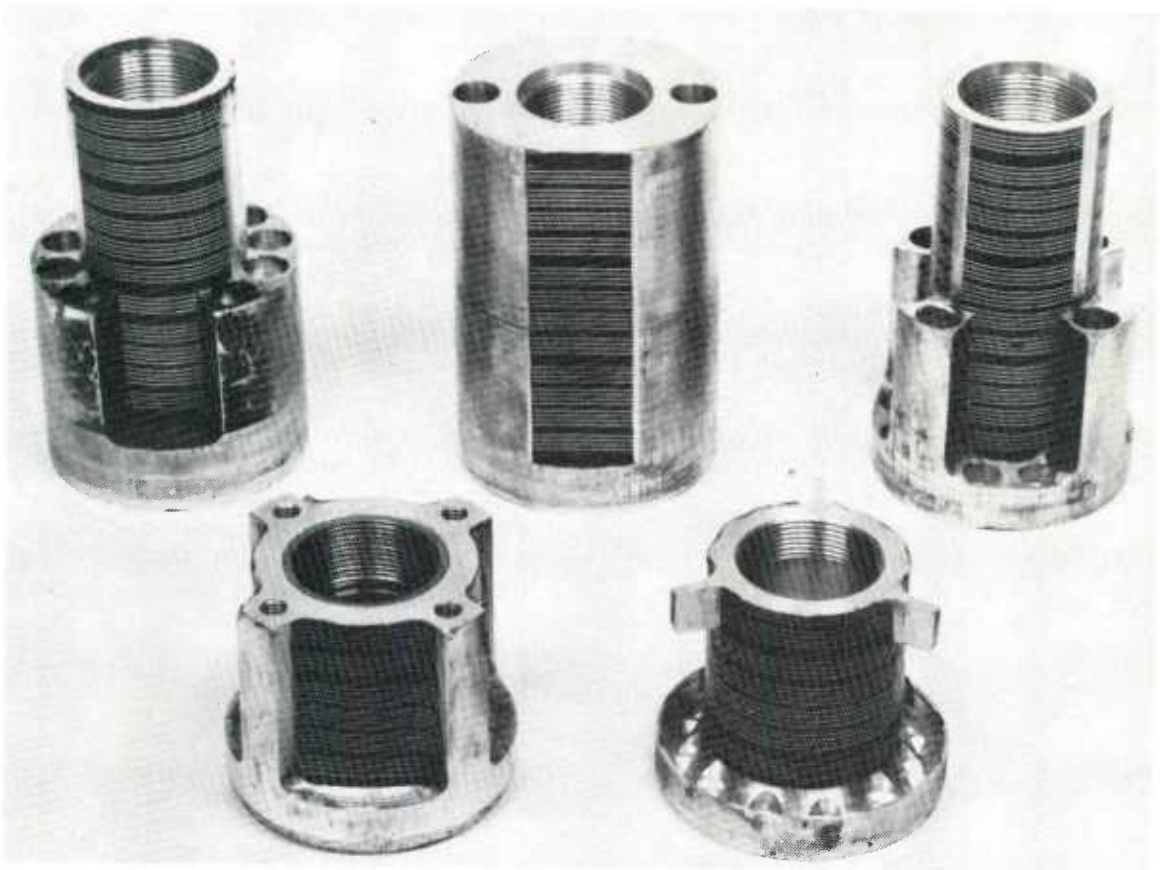


Figure 20. Typical projectiles used in study.

Each set of test data is plotted along with predictions from each of the three theories (fig. 21 to 23). The theoretical predictions were obtained from the program THEORY.

If the gun behaved as a constant acceleration gun, then the velocity would be proportional to the square root of the acceleration. Therefore, the x-axis in the figures was chosen to be the square root of the initial acceleration rather than the initial acceleration. This choice linearizes the results somewhat and eliminates a considerable amount of curvature from the plots.

The first set of results (fig. 21) represents the data for a nitrogen pressure

driver, with the projectile restraining section between gun sections 1 and 2. This is a long barrel, short driver case with a relative pin position of 0.117. Each plus sign represents an experimental result. Except for an occasional "low" point, the data follow Pidduck-Kent theory. Referring to figure 17, a vertical line drawn at a relative pin position of 0.117 would intersect only Pidduck-Kent curves, regardless of initial acceleration.

Figure 22 shows the results obtained with a nitrogen driver and the projectile restraining section between gun sections 3 and 4. The relative pin position is 0.372. At initial accelerations up to about 1000 g, Pidduck-Kent is favored; above 1000 g, the results are closest to and within a few percentage points

of infinite driver theory. Again referring to figure 17, a vertical line at 0.372 would intersect Pidduck-Kent curves up to 800 g and infinite driver curves for higher initial accelerations.

The third set of data (fig. 23) was based on a configuration that was geometrically the same as the last set, but the driver gas used was helium. In this set, Lagrange theory or Pidduck-Kent theory applies equally to the data below 1000 g. At higher acceleration, the data slightly exceed Pidduck-Kent theory. Reference to figure 16 at a relative pin position of 0.372 yields only Pidduck-Kent intersections up to initial accelerations of 3000 g. Figure 18 shows intersections with infinite driver only above 9000 g.

The combined experimental results are presented in figure 24 for comparison. The data of each set (delta, plus, and times sym-

bols) are sufficiently separated from those of the other sets so that it is relatively easy to picture the trends associated with each. Experimental results at about 200-g initial acceleration show equivalent or slightly higher velocities for nitrogen compared with those for helium at the same projectile restraining section. As the initial acceleration increases, the superior results for helium are apparent.

The experimental velocities attained with nitrogen at a relative pin position of 0.372 exceed those at a relative pin position of 0.117 (fig. 24). However, the difference in velocities decreases with increasing initial acceleration. These two sets of data should eventually cross when the 0.117 gun acceleration is sufficiently large for infinite driver theory to govern. Then for larger initial acceleration, the longer barrel of the gun with a relative pin position of 0.117 (0.883) produces higher velocities than that with 0.372 (0.628).

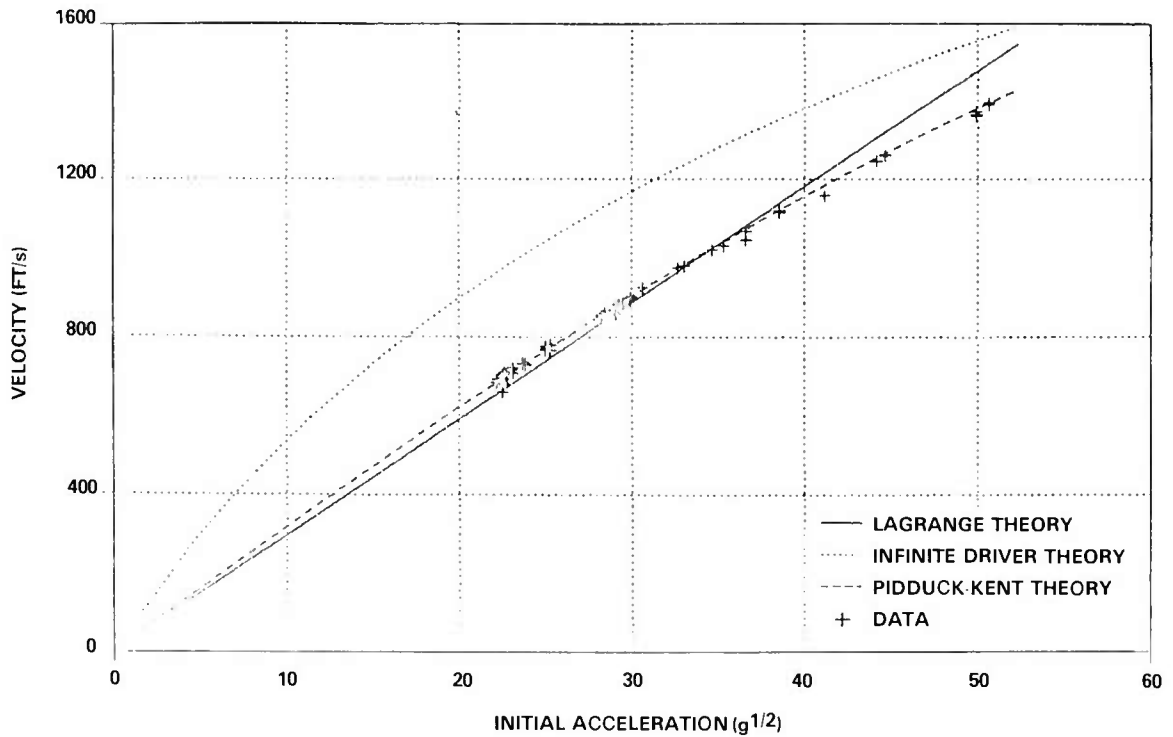


Figure 21. Data and theories, nitrogen, relative pin position of 0.117.

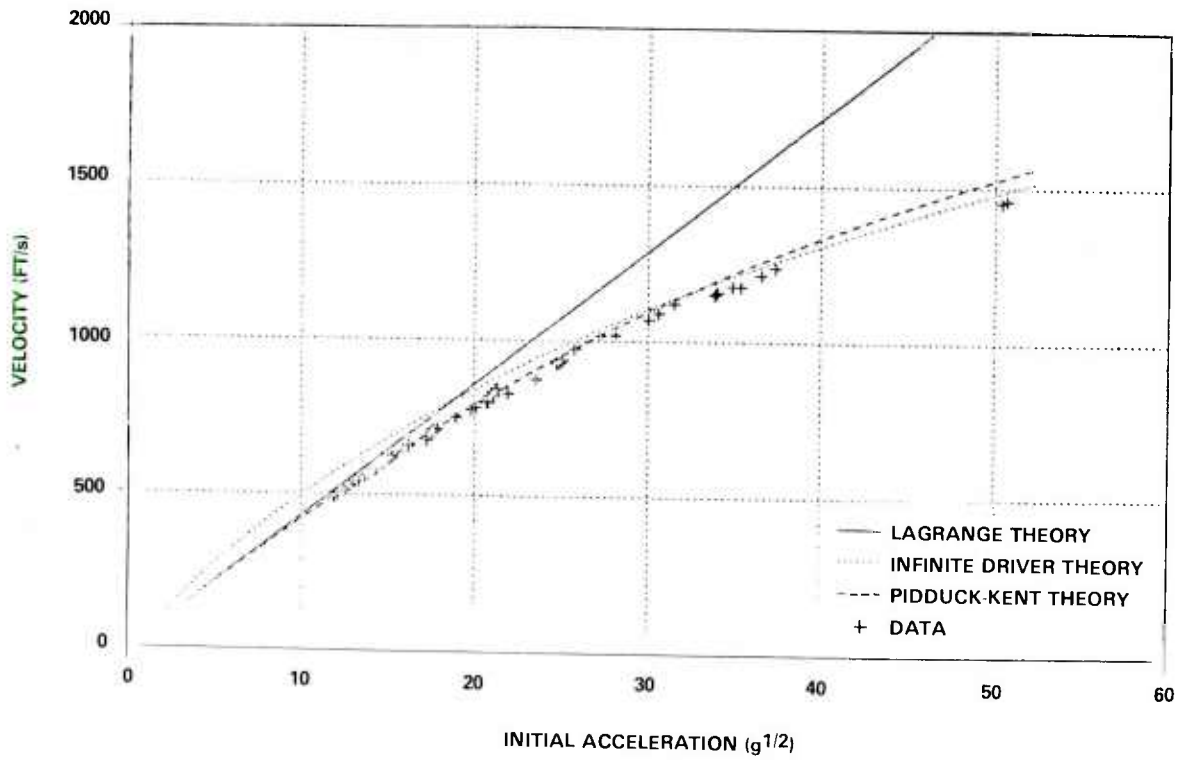


Figure 22. Data and theories, nitrogen, relative pin position of 0.372.

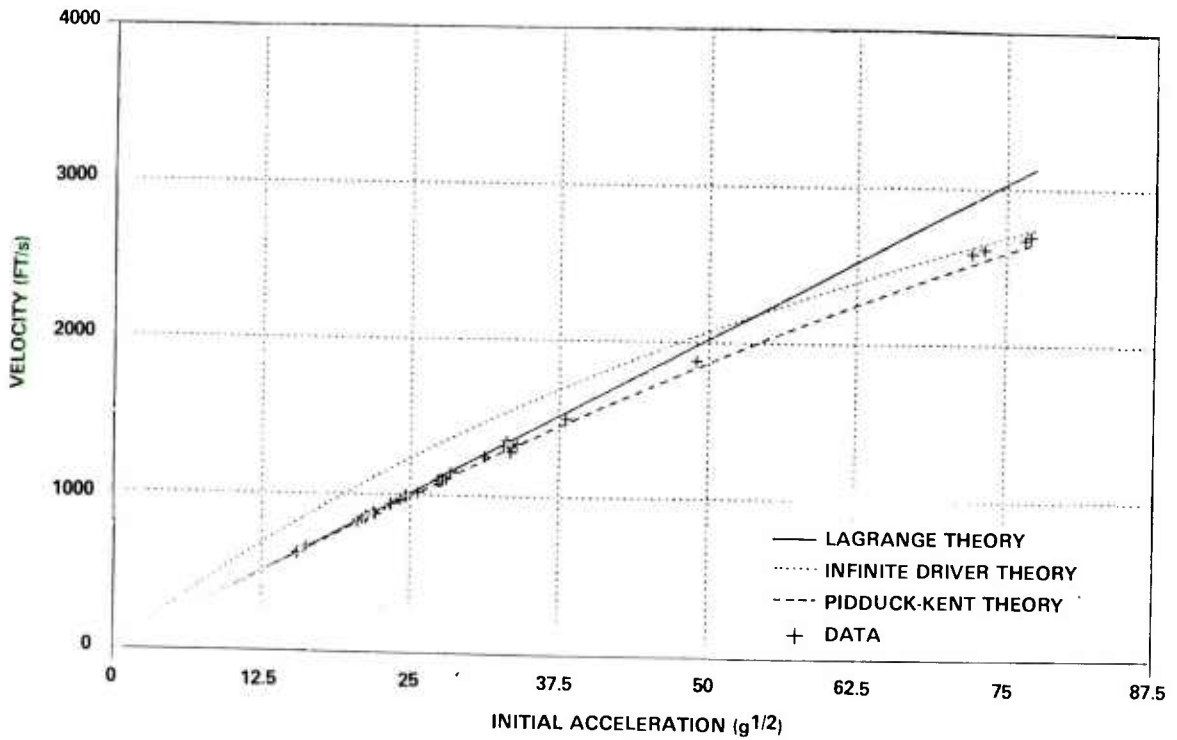


Figure 23. Data and theories, helium, relative pin position of 0.372.

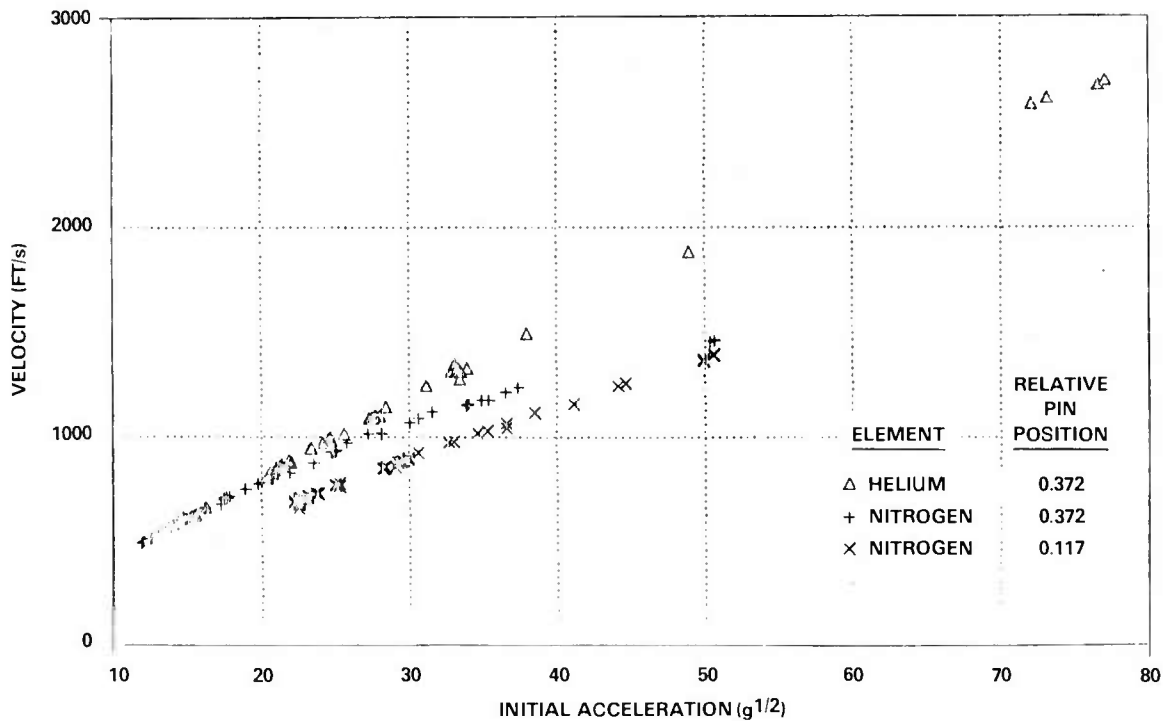


Figure 24. Combined data, nitrogen and helium, relative pin positions of 0.117 and 0.372.

7. DISCUSSION

The experimental data for helium show higher velocities than for nitrogen at the same initial acceleration for velocities above 750 ft/s (225 m/s) (fig. 24). Helium follows Lagrange theory or Pidduck-Kent theory for the lower velocities and Pidduck-Kent for velocities above 1500 ft/s (500 m/s) (fig. 23). From the assumptions associated with these theories, it follows that as the projectile restraining section is moved away from the muzzle (decreasing relative pin position), these theories should continue to be appropriate. (It is not clear that these theories will be appropriate if the pin is shifted *toward* the muzzle. In this region, the gun approaches an infinite driver configuration, and infinite driver theory may govern.)

No clear-cut optimum relative pin position has resulted from the experimental or

theoretical investigation. Choice of relative pin position becomes a matter of judgment, which may be influenced by secondary considerations such as (1) expected range of use, (2) minimization of the volume (mass) of gas used, (3) minimization of the pressure behind the projectile at muzzle exit, and (4) ease of operation.

Figure 25 presents Lagrange curves and Pidduck-Kent curves for the first three possible projectile restraining section locations in the gun. Analysis of Lagrange theory readily yields the initial acceleration required at one projectile restraining section compared with the acceleration at a second projectile restraining section (acceleration ratio) to produce the same muzzle velocity. (From the form of the Pidduck-Kent equations, it appears that this acceleration ratio varies with the initial acceleration. Nevertheless, from the Lagrange curve and Pidduck-Kent curve shapes of figure 25,

any generalization based on trends applicable to Lagrange theory is generally applicable to Pidduck-Kent theory in this acceleration regime.)

Consider the quantity (mass) of gas used per test as a secondary criterion. As an example, assume that helium is used and that the gun is operating in the Lagrange regime or the Pidduck-Kent regime. (If the acceleration is sufficiently high to operate in the infinite driver regime, then reducing the relative pin position increases muzzle velocity.) The gas required for a shot is immediately available from the gun dimensions and required initial pressure (acceleration) (program WTVSRPP).

(relative to the projectile mass) required versus projectile restraining section location (relative pin position) for velocities from 300 to 1000 ft/s (90 to 300 m/s). A significant observation is that the minimum gas mass required is at a relative pin position of zero. (Although as the relative pin position approaches zero the required pressure becomes infinite, the volume and the mass go to zero.) The quantity of gas increases monotonically with increasing relative pin position. It is apparent that to minimize the amount of gas used, one would choose the smallest driver, whether it be helium (fig. 26) or nitrogen (fig. 27). This choice is important when the gun operates with helium because of the time involved in purging and filling the driver; that is, the smaller the driver and the less gas required, the faster the operation (firing).

Figures 26 and 27 show the mass of gas

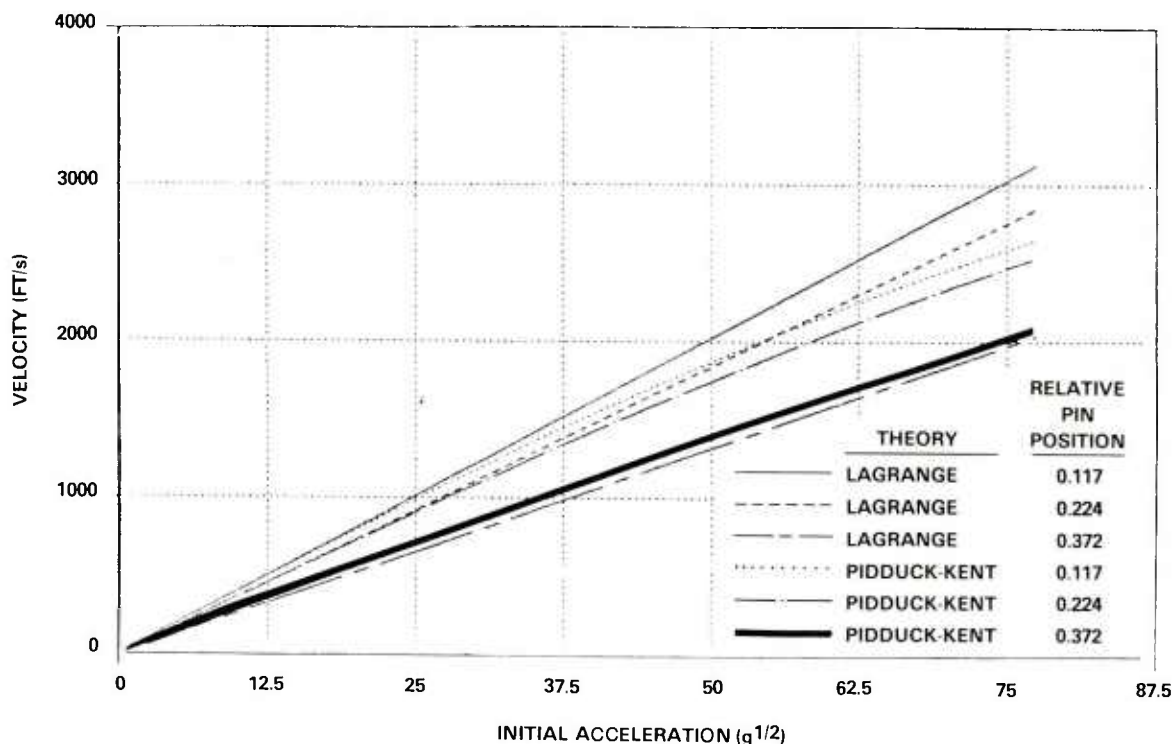


Figure 25. Lagrange and Pidduck-Kent theories, helium, relative pin positions of 0.117, 0.244, and 0.372.

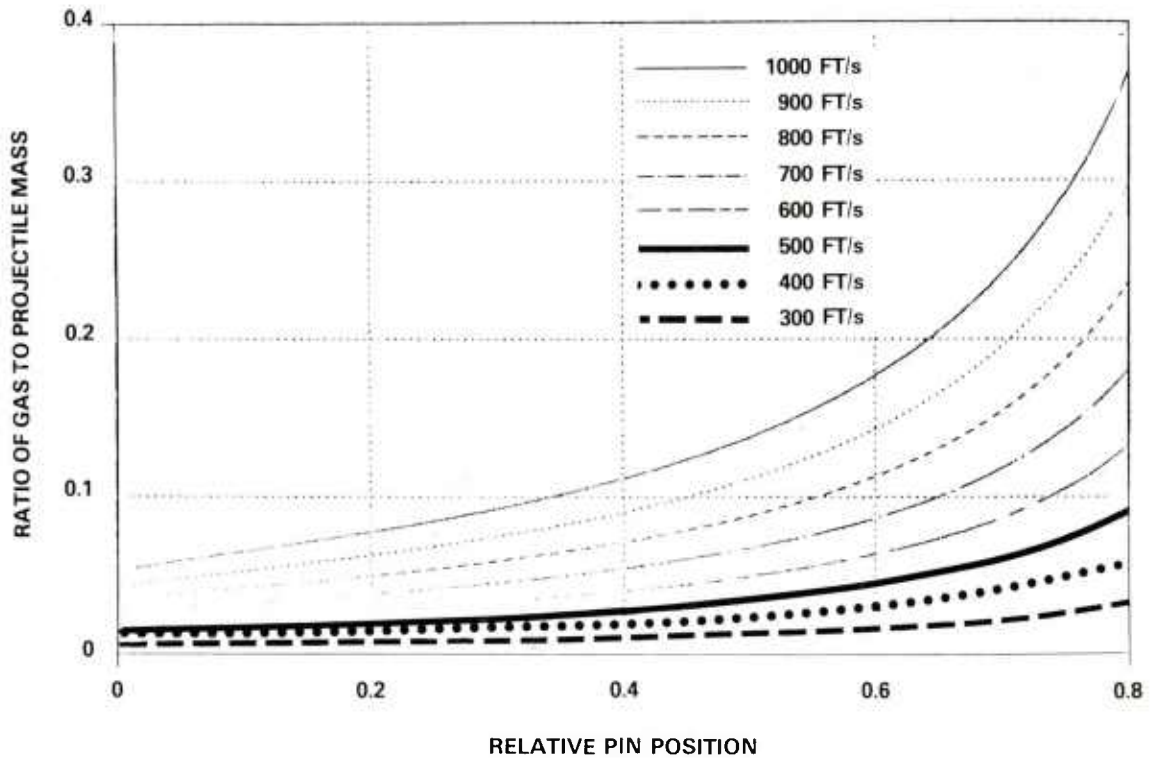


Figure 26. Helium (Lagrange theory).

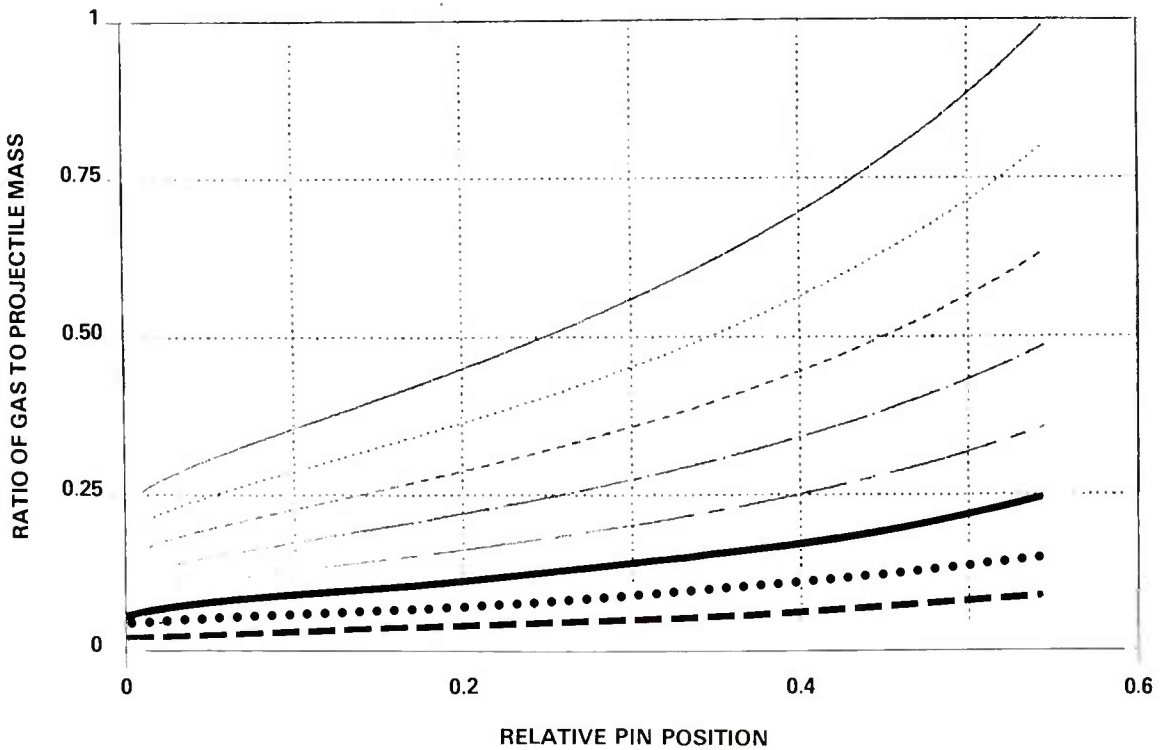


Figure 27. Nitrogen (Lagrange theory).

Another consideration is the amount of energy stored in the gas when the projectile exits from the gun. This energy may damage equipment or interfere with the impact experiments. The theory presented in program NRG predicts the energy transferred from the driver gas to the projectile. The final driver gas energy (enthalpy) was obtained by subtracting the projectile muzzle energy from the initial gas energy. The results (fig. 28) show that the smaller the driver (small relative pin position), the greater the amount of energy transferred to the projectile and therefore the smaller the amount of energy remaining in the gas. (Note: The greater the degrees of freedom of the gas, the less energy transferred to the projectile.)

The effects of projectile restraining section placement on several parameters are presented in table 2. The five projectile restraining section locations are nearest the

breech. The effects are arbitrarily referenced to the present pin location (between sections 3 and 4). For example, the initial acceleration required to produce a particular muzzle velocity for a projectile restraining section between sections 3 and 4 is assigned a value of 1. The initial acceleration required to produce the same velocity for the pin between sections 1 and 2 would be 2.02 times as great (table 2, second parameter).

Note that the fourth pin position (relative pin position of 0.500) is close to the optimum location based on Lagrange theory (relative pin position of 0.465, helium, table 1). Table 2 indicates that moving the pin to this position reduces the initial acceleration by only 3 percent, but requires 30 percent more gas. If a 21-percent increase in initial acceleration is tolerable, then gas usage can be cut 21 percent by moving the projectile restraining sec-

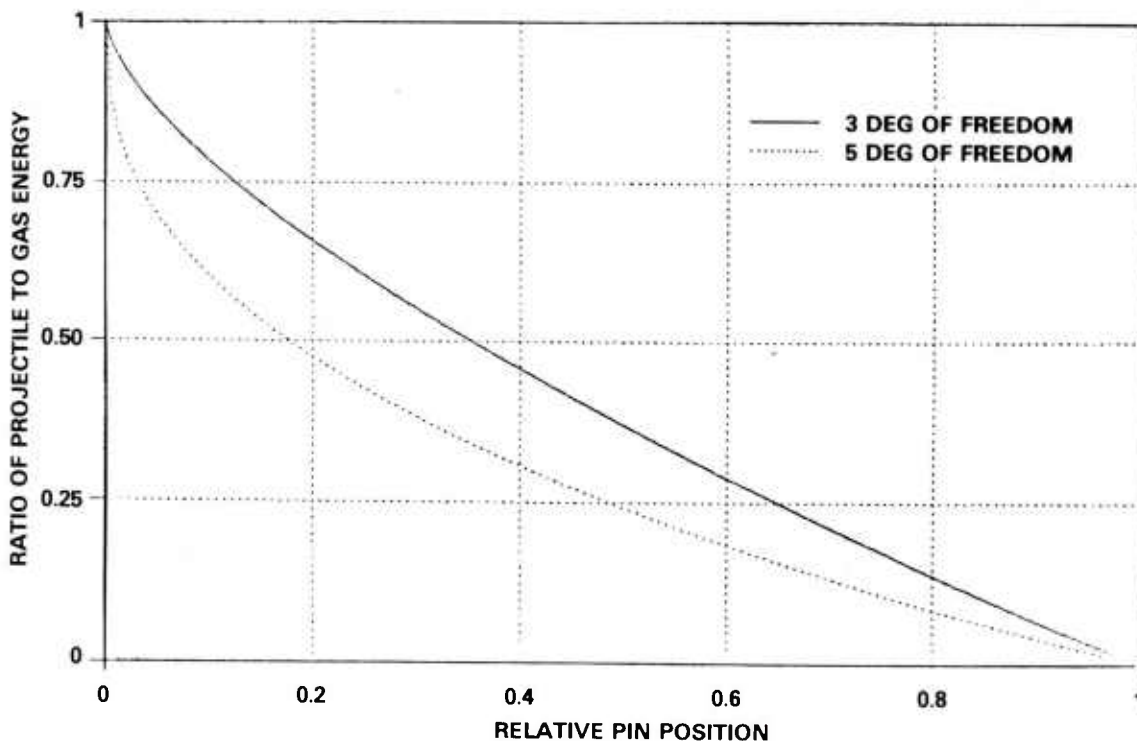


Figure 28. Fraction of energy gained by projectile from driver gas (Lagrange theory).

tion back one section. Moving the projectile restraining section back two sections to its original location is not as attractive.

The data of the fourth parameter of table 2 show that there would be a 24-percent reduction in gas energy at breech exit by moving the projectile restraining section back one section.

8. CONCLUSIONS AND REMARKS

A theoretical and experimental study was made to determine where a fixed length, constant diameter tube should be divided so that when one portion of the tube is used as a gun barrel and the remaining portion is used as a driver (chamber), the maximum projectile muzzle velocity can be obtained for a particular driver gas and initial acceleration. Studies of several HDL 4-in. gas gun configurations indicated that no one configuration is best for all velocities greater than 500 ft/s (150 m/s), the range of interest. (For lower velocities, the gun is used in the 1-atm configuration, where the entire tube length is used for the barrel and ambient air is the driver. This configuration is not relevant to the study.)

For the cases considered (short driver versus long driver, helium versus nitrogen, and in-

itial accelerations from 200 to 5000 g), the experimental results generally follow different theories in different operating regimes: Lagrange theory in the very low Mach number regime, infinite driver theory in the very high acceleration regime, and Pidduck-Kent theory in other regimes. Although the various results are explicable in terms of the postulated theories, no one theory accurately predicts the entire range of experimental results. Therefore, it is difficult to infer a general optimum configuration because theoretical predictions around the optimum are in most cases conflicting or poorly defined or both. Because no clear-cut division of the tube can be inferred to optimize muzzle velocity, secondary criteria such as quantity of gas required, energy remaining in the gas, (statistical) utilization of the gun, and ease of operation can influence the configuration without unduly affecting muzzle velocity.

In conclusion, significant savings in gas usage and reductions in gas energy at muzzle exit can be realized with only a small acceleration penalty by reconfiguring the HDL gun to a one section longer barrel and one section shorter driver.

TABLE 2. EFFECTS OF PROJECTILE RESTRAINING SECTION WITH HELIUM DRIVER

Parameter	Projectile restraining section between sections				
	1 and 2	2 and 3	3 and 4	4 and 5	5 and 6
Relative gun position	0.117	0.245	0.372	0.500	0.628
Initial acceleration ratio to attain particular muzzle velocity	2.02	1.21	1.00	0.97	1.07
Ratio of required gas masses	0.63	0.79	1.00	1.30	1.81
Ratio of final driver gas energy to initial gas energy	0.46	0.76	1.00	1.22	1.42

Note: Reference is projectile restraining section location between sections 3 and 4; relative pin position is 0.372.

ACKNOWLEDGEMENT

The author extends his appreciation to Forrest Nelson, William McIntosh, and Robert Kayser for their careful performance of the experiments and their faithful recording of the data.

NOMENCLATURE

a	Initial sound speed in driver reservoir
A	Cross-sectional area of gun barrel
K	$2APL/M$
L	Total gun length (barrel + driver, $X + Y$)
M	Projectile mass
N	Degrees of freedom of driver gas: $N = 3$ for helium and $N = 5$ for nitrogen
P	Initial gas pressure in driver reservoir
u	Projectile Mach number based on initial sound speed (U/a)
U	Projectile velocity
X	Length of driver reservoir
Y	Length of barrel
Z	Relative pin position in gun (X/L)

APPENDIX A.—COMPUTER PROGRAMS

The computer programs on the following pages were written in VAX-11 BASIC, a compilable, native mode language. Since the language is not as restrictive as standard BASIC, a few of the major differences are pointed out for those who wish to adapt the programs to their own computers.

Comments.—In addition to REM statements, the exclamation point serves as a comment delimiter.

Example:

```
10 THIS=THAT ! SET THE VARIABLE THIS EQUAL TO THE VARIABLE THAT
```

Continuations.—A line may be continued on the following line when the last character on the line to be continued is an ampersand.

Example:

```
10 THIS= &  
THAT
```

Multiline statements.—Statements to be executed in continuous order can be on the same line if they are separated by a back slash.

Example:

```
10 THIS=7 \ THAT=8
```

Line numbers.—Line numbers are not required if the statement is indented at least one character from the left margin. Lines are executed in order.

Example:

```
10 THIS=7  
    THAT=8
```

Variables.—Names of variables are allowed to be up to 32 characters long and can contain most ASCII characters.

Example:

```
10 DEG.OF.FREEDOM=2/(GAMMA-1) \ AVG__A=SUM__A/NO
```

Use of these enhancements produced programs that are easier to read and to follow than standard BASIC. The programs included are PDQ, CONDIA, CATCH, CDPK, THEORY, WTVSRPP, and NRG. Other programs were used to retrieve the data from the record files and to plot the data. However, these are not general and would be of little use to the reader.

APPENDIX A

10 REM

PDQ.BAS
APRIL 1980 H D CURCHACK

PIDDUCK-KENT THEORY
SEE SEIGEL HIGH SPEED GUNS PP 180-181

```
20      DIAM=4 ! IN. !           \      X SEC AREA=PI*DIAM^2/4
      GAS$(1)="HELIUM"         \      GAS$(2)="NITROGEN"
      SPEED(1)=3300 !FT/SEC!   \      SPEED(2)=1145
      DEG OF FREEDOM(1)=3      \      DEG OF FREEDOM(2)=5
      GAMMA(T)=5/3             \      GAMMA(Z)=7/5
      THISSTEP=500             \      DEL_MU=1/THISSTEP
      TOTAL_L=94
```

```
30 FOR GAS=1 TO 2 \ PRINT GAS$(GAS)
      NO=DEG OF FREEDOM(GAS) \ CO=SPEED(GAS) \ GAMA=GAMMA(GAS)
      GAS_STEP(T)=.0005 \ GAS_STEP(2)=.0025
```

```
      OPEN "PK."+LEFT$(GAS$(GAS),3) FOR OUTPUT AS FILE #GAS
      GOSUB 40 FOR INIT_G=200 TO 3001 STEP 200
      CLOSE #GAS
```

```
      OPEN "PK1."+LEFT$(GAS$(GAS),3) FOR OUTPUT AS FILE #GAS
      GOSUB 40 FOR INIT_G=3000 TO 15001 STEP 3000
      CLOSE #GAS
```

```
      NEXT GAS
      GOTO 90
```

```
40      ! PIDDUCK KENT SUBROUTINE
      ! -----
```

```
      PRINT "INITIAL G =";INIT_G      ! TERMINAL OUTPUT DURING RUN
```

```
      ! A_BAR IS A PARAMETER USED BY SIEGEL TO DETERMINE G/M AND VEL
```

```
      FOR A_BAR=0 TO 1 STEP GAS_STEP(GAS)
```

```
        ! INTEGRATION !
```

```
        INTEGRAL=0
```

```
        FOR INTVAL=0 TO THISSTEP
```

```
          MU=INTVAL*DEL MU
```

```
          F OF MU=(1-A_BAR*MU^2)^(NO/2)
```

```
          INTGRAL=INTGRAL+F_OF_MU
```

```
        NEXT INTVAL
```

```
        INTEGRAL=(INTEGRAL-.5*(1+F_OF_MU))/THISSTEP
```

```
        ! INTEGRATION COMPLETE !
```

```
        G_OVER_M=(NO+2)*A_BAR*(1-A_BAR)^(-(NO+2)/2)*INTGRAL
```

```
50 REM      - CONVERT G OVER M TO INIT_G AND DRIVER_L -
      CO^Z=GAMA*P/RHO
```

```
      ALSO
```

```
      WEIGHT OF GAS (G)=RHO*X_SEC_A*DRIVER_L
```

```

                                DRIVER_L=G/(RHO*X SEC A)
                                =G*CO^2/(GAMA*P*X SEC A)
                                =G*CO^2/(GAMA*INIT_G*M)
60  THEREFORE
                                DRIVER L=G OVER M*CO^2/(GAMA*INIT_G*32.2)
                                FRAC L=DRIVER_L/TOTAL_L
                                IF FRAC L>1
                                    THEN 80
                                ELSE VEL=2*CO/(GAMA-1)*SQR(A BAR*(1-FRAC L^(GAMA-1)))
                                PRINT #GAS USING "#.###F , ###.#",FRAC_L,VEL
70  NEXT A BAR
80  PRINT #GAS," 1. , 0."
    RETURN
90  END

```



```

RETURN
60 ! DETERMINE VELOCITY WHICH IS IMPLICIT IN TERMS OF DISTANCE.
      YO=G X L*INIT_G/CO^2
      F1=.T
      UO=SQR(YO)
      IF UO>NO-1
70         THEN UO=NO-1
      X9=Y0/10000
      IF X9<1.00000E-07
          THEN X9=1.00000E-07
80     !                               LOOP
      X=NO1 * (1- (1-UO/NO1) / (1-UO/NO)^N1 )
      X1=Y0-X
      SO=SGN(X1)
      IF F2=S0
          THEN F2=0
              F1=F1/2
90     IF ABS(X1)>X9
          THEN UO=(1+S0*(F1))*UO
              F2=-S0
              GO TO 80
      !                               END LOOP
100    RETURN
110 END

```



```

50   FOR DRVR=D START TO TOTAL L STEP D_STEP
      X_D=32.2*DRVR*INIT_G/C_SQ
      !
      REFLECTED WAVE COMPUTATION
      F=(1+F1_GAM*X_D)^(-2*F2_GAM/F1_GAM)
      X_B=((1-F1_GAM*F)/F^(F1_GAM/F2_GAM)/F2_GAM+1)/F1_GAM
      IF INIT_G<>0
      THEN TRAVEL_D=X_B*C_SQ/(32.2*INIT_G)
60   CATCH D=(TRAVEL_D+DRVR)/TOTAL_L
      IF CATCH_D<1
      THEN 70
      ELSE RESOLUTE=RESOLUTE+1
      IF RESOLUTE>10
      THEN 80
      ELSE D_START=DRVR-D_STEP
      D_STEP=D_STEP/2
      GOTO 50
70   NEXT DRVR
80   FRAC_L=DRVR/TOTAL_L
      PRINT #FYL USING "###.### , ###.### , ####", FRAC_L,CATCH_D,INIT_G
      RETURN
90 END

```



```

90 RETURN

100 ! SUBROUTINE TO DETERMINE VELOCITY GIVEN A LENGTH FOR
! A CONSTANT DIAMETER GUN. EQUATION IS
! IMPLICIT FOR VELOCITY AND EXPLICIT FOR LENGTH.

Y0=G_X_L*INIT_G/SPEED(GAS)^2
!-----

110 F1=.1
I=0
U0=SQR(Y0)
IF U0>N0-1
THEN U0=N0-1
120 X9=Y0/10000
IF X9<1.00000E-07
THEN X9=1.00000E-07

! LOOP

130 X=(N0/(N0+1)) * (1- (1-(N0+1)*U0/N0) / (1-U0/N0)^(N0+1) )
!-----

! SEE FOR EXAMPLE EQ B11, HDL-TR-1330, CURCHACK
X1=Y0-X
S0=SGN(X1)
140 IF F2=S0
THEN F2=0
F1=F1/2
150 IF ABS(X1)>X9
THEN U0=(1+S0*(F1))*U0
F2=-S0
GO TO 130

160 RETURN ! END LOOP

170 END

```

APPENDIX A

10 REM

THEORY

APR 80

H D CURCHACK

Range of Speeds vs Initial g for Isentropic Gun.
 Same for Constant Dia Gun. Outputs to separate files.
 Same for Pidduck-Kent Gun. Outputs to separate files.

CONSTANTS

```
20     DIA=4 ! IN.           \       X_SEC_AREA=PI*DIA^2/4
SHORT=1                     \ REG=2                \ LNG=3
BARREL(SHORT)=59           \ BARREL(REG)=71          \ BARREL(LNG)=83
DRIVER(SHORT)=35          \ DRIVER(REG)=23        \ DRIVER(LNG)=11
RPP$(SHORT)=" .SHO;1"   \ RPP$(REG)=" .MED;1"       \ RPP$(LNG)=" .LON;1"
G X L(I)=32.2*BARREL(I)  ! (FT/SEC)SQUARE
FOR I=SHORT TO LNG
TOTAL_L=BARREL(SHORT)+DRIVER(SHORT)

    GAS$(1)="HELIUM"        \       GAS$(2)="NITROGEN"
    SPEED(1)=3300 !FT/SEC!  \       SPEED(2)=1145
    DEG OF FREEDOM(1)=3     \       DEG OF FREEDOM(2)=5
    GAMMA(T)=5/3           \       GAMMA(Z)=7/5
    FORM2$="#.### , #.###" \       FORM4$="####.# , ####.#"
    MAX_INIT_G(1)=5500     \       MAX_INIT_G(2)=2750
```

```
30 FOR GAS=1 TO 2
  START G=MAX_INIT_G(GAS)
  CU=SPEED(GAS)
  GAMA=GAMMA(GAS)
  NO=DEG OF FREEDOM(GAS)
  MORE$=LEFT$(GAS$(GAS),3)
  FOR BARREL%=SHORT TO LNG
    MY_FILES="ISEN"+MORE$+RPP$(BARREL%)
    OPEN MY_FILES FOR OUTPUT AS FILE #GAS
    PRINT "FILE=";MY_FILES
    FRAC_L=DRIVER(BARREL%)/TOTAL_L
    FRAC_L_FACT=NO*FRAC_L*(1-FRAC_L^(2/NO))/2
    FOR INIT_G=0 TO START_G STEP 25
      YO=G X L(BARREL%)*INIT_G/CO^2
      K=2*(YO*TOTAL_L/BARREL(BARREL%))
      MACH_NO=SQR(K*FRAC_L_FACT)
      PRINT #GAS USING FORM4$,INIT_G,MACH_NO*CO
    NEXT INIT_G
  CLOSE #GAS
```

```
40
  MY_FILES="COND"+MORE$+RPP$(BARREL%)
  OPEN MY_FILES FOR OUTPUT AS FILE #GAS
  PRINT "FILE=";MY_FILES
  G X L(3)=G X L(BARREL%) ! (FT/SEC)SQUARE !
50 FOR INIT_G=0 TO START_G STEP 25
  RO=INIT_G/X_SEC_AREA
  GOSUB 170
  PRINT #GAS USING FORM4$, INIT_G , UO*CO
```

```

60         NEXT INIT G
           CLOSE #GAS
70     NEXT BARREL%

!           PIDDUCK-KENT THEORY
!           SEE SEIGEL HIGH SPEED GUNS PP 180-181

OPEN "PIDD"+MORE$+RPP$(BRL%) FOR OUTPUT AS FILE #BRL% &
                                           FOR BRL%=1 TO 3
FLAG(1)=0 \ FLAG(2)=0 \ FLAG(3)=0
GAS STEP(1)=.001 \ GAS_STEP(2)=.0025
GOSUB 100
CLOSE #1,#2,#3
80 NEXT GAS
90 GOTO 220

100 ! A BAR IS A PARAMETER USED BY SEIGEL TO DETERMINE G/M AND VEL !
THISSTEP=500 \ DEL MU=1/THISSTEP
FOR A BAR=0 TO 1 STEP GAS STEP(GAS)
FOR SUMMY=0 TO THISSTEP-1
    MU=SUMMY*DEL MU
    NUMU=MU+DEL MU
    DEL AREA=(1-A BAR*MU^2)^(NO/2)+(1-A BAR*NUMU^2)^(NO/2)
    INTGRAL=INTGRAL+DEL AREA
NEXT SUMMY
INTGRAL=INTGRAL/2/THISSTEP
G OVER M=(NO+2)*A BAR*(1-A BAR)^(-(NO+2)/2)*INTGRAL
110 FOR BRL%=SHORT TO LNG
    FRAC L=DRIVER(BRL%)/TOTAL_L
    IF FLAG(BRL%)=1
        THEN 130 ! DON'T COMPUTE IF INIT G TOO LARGE
    INIT G(BRL%) = G OVER M*CO^2/(GAMA*DRIVER L*32.2)
    VEL=2*CO/(GAMA-1)*SQR(A BAR*(1-FRAC L^(GAMA-1)))
    IF INIT G(BRL%)<=START G
        THEN PRINT #BRL% USING FORM4$,INIT_G(BRL%),VEL
        ELSE FLAG(BRL%)=1
120
130     NEXT BRL%
140     INTGRAL=0
    IF INIT_G(1)>START_G
        THEN RETURN
150 NEXT A BAR
160 RETURN

!           COMPUTATION SUBROUTINE

170     YO=G X L(3)*RO*X_SEC_AREA/CO^2
    F1=.I
    I=0
    UO=SQR(YO)
    IF UO>NO-1
        THEN UO=NO-1
180     X9=YO/10000
    IF X9<1.00000E-07
        THEN X9=1.00000E-07

```

APPENDIX A

```
!                                LOOP
190  X=(NO/(NO+1)) * (1- (1-(NO+1)*UO/NO) / (1-UO/NO)^(NO+1) )
      I=I+1
      X1=Y0-X
      SO=SGN(X1)
      IF F2=SO
        THEN F2=0
            F1=F1/2
200  IF ABS(X1)>X9
      THEN UG=(1+SU*(F1))*UG
            F2=-SO
            GO TO 190
!                                END LOOP
210 RETURN
220 END
```


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